

Exam CT4150 Plastic Analysis of Structures
Thursday 17 January 2008, 9:00 – 12:00 hours

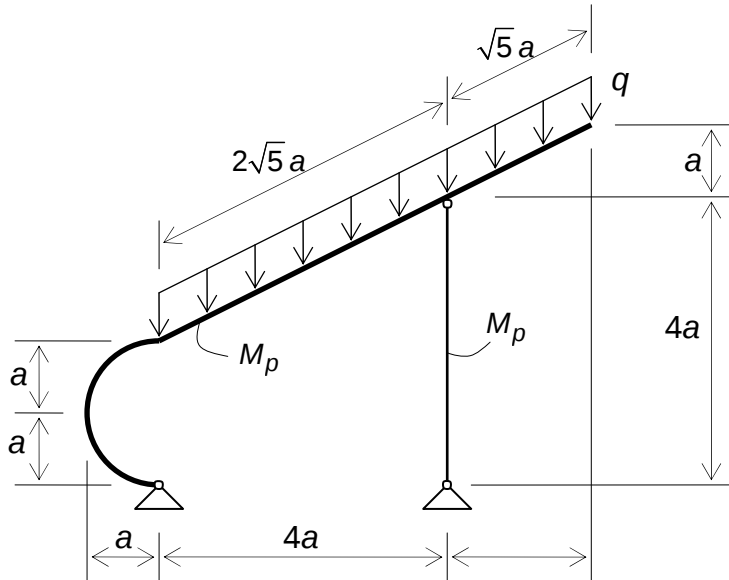


Figure 1. Curved frame

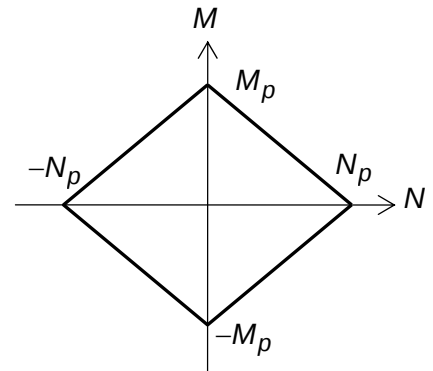


Figure 2. Yield contour

Problem 1

A frame consists of a curved beam and a column with strengths M_p (Fig. 1). The beam is connected with hinges to the foundation and to the column. The beam is loaded by an evenly distributed force q . The following relation exists between the plastic moment M_p and the plastic normal force N_p (Fig. 2).

$$N_p = \beta \frac{M_p}{a}$$

The influence of shear on the yield contour is neglected. Buckling and second order effects are not considered.

- a** Assume $\beta \rightarrow \infty$. Determine the collapse load q for all possible mechanisms. Write the collapse loads as functions of M_p and a . What is the decisive collapse load? (1.5 points)
- b** Assume $\beta \rightarrow \infty$. Draw the bending moment diagram and normal force diagram for the structure at the moment of collapse. (1 point)
- c** Assume $\beta = 30\sqrt{5}$. Choose one of the following problems (You need not do both).
 - Determine the largest lower-bound for q .
 - Determine the smallest upper-bound for q .
If you choose the upper-bound you only need to write down the equations and not solve the equations (2 points).

Problem 2

A square plate is supported on all edges except for a length of $a/2$ (Fig. 3). The support is special in that it resists only compression and no tension. Neither it resists moments (Fig. 4). The plate carries a point load F in the middle. The plate is isotropic and homogeneous, therefore $m_{px} = m_{py} = m'_{px} = m'_{py} = m_p$.

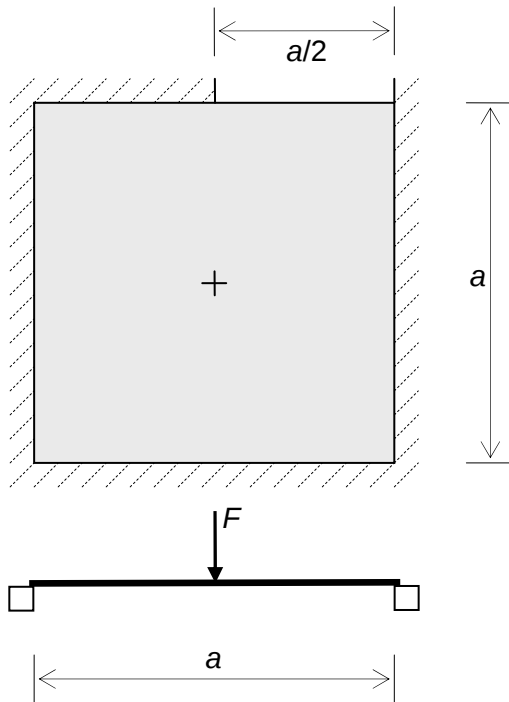


Figure 3. Plate dimensions and loading

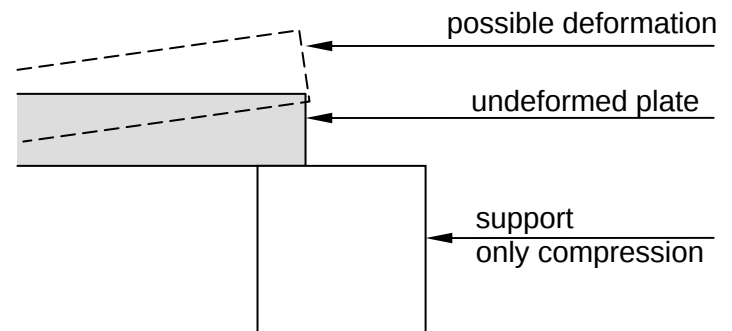


Figure 4. Plate support

- a** Consider the yield line patterns of Figure 5. Which of these patterns give kinematically possible mechanisms. (1 point)

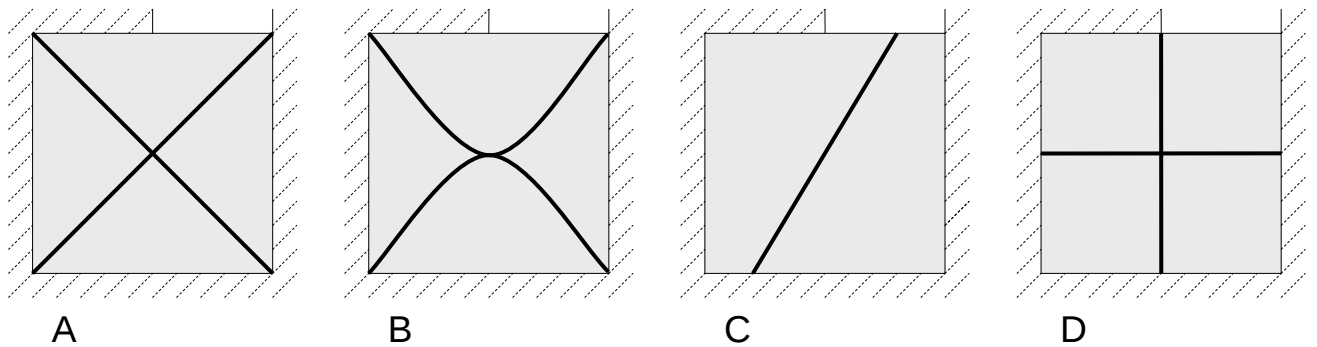


Figure 5. Yield line patterns of problem 2a

- b** Consider the yield line pattern of Figure 6. Determine an upper bound for F expressed in m_p (1.5 points).

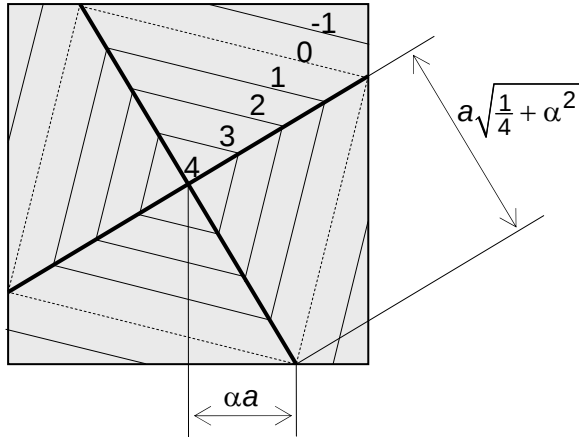


Figure 6. Yield line pattern of problem 2b

- c Determine the largest lower-bound for F using torsion free beams ($m_{xy} = 0$) in the x direction and y direction. Assume that the point load is distributed over an area with a width $a/10$. (1.5 points)

Problem 3

- a When we design reinforcement according to

$$\begin{aligned} m_{px} &= m_{xx} + |m_{xy}| \\ m_{py} &= m_{yy} + |m_{xy}| \\ m'_{px} &= -m_{xx} + |m_{xy}| \\ m'_{py} &= -m_{yy} + |m_{xy}| \end{aligned}$$

what will be the crack direction (angle α)? (0.5 point)

- b In elastic analysis a point load F on a plate gives unrealistic large moments directly under the point load. Designers cut these moments at a value $F/3$. Is this safe for a mid load, a freed edge load and a corner load? Assume that $m_{px} = m_{py} = m'_{px} = m'_{py} = m_p$. (0.5 point)

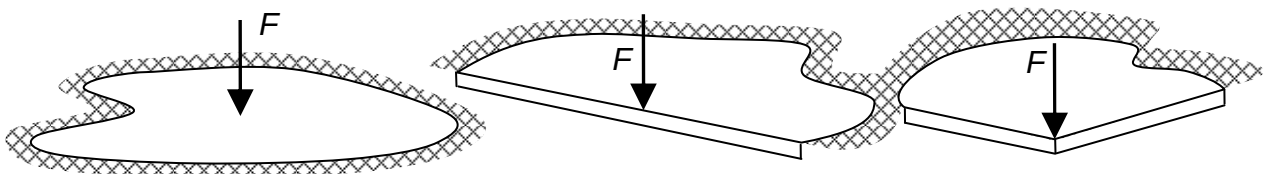


Figure 7. Point loads at three locations

- c Which of the following reinforcement detailing of a column bracket will provide the largest strength? (0.5 point)

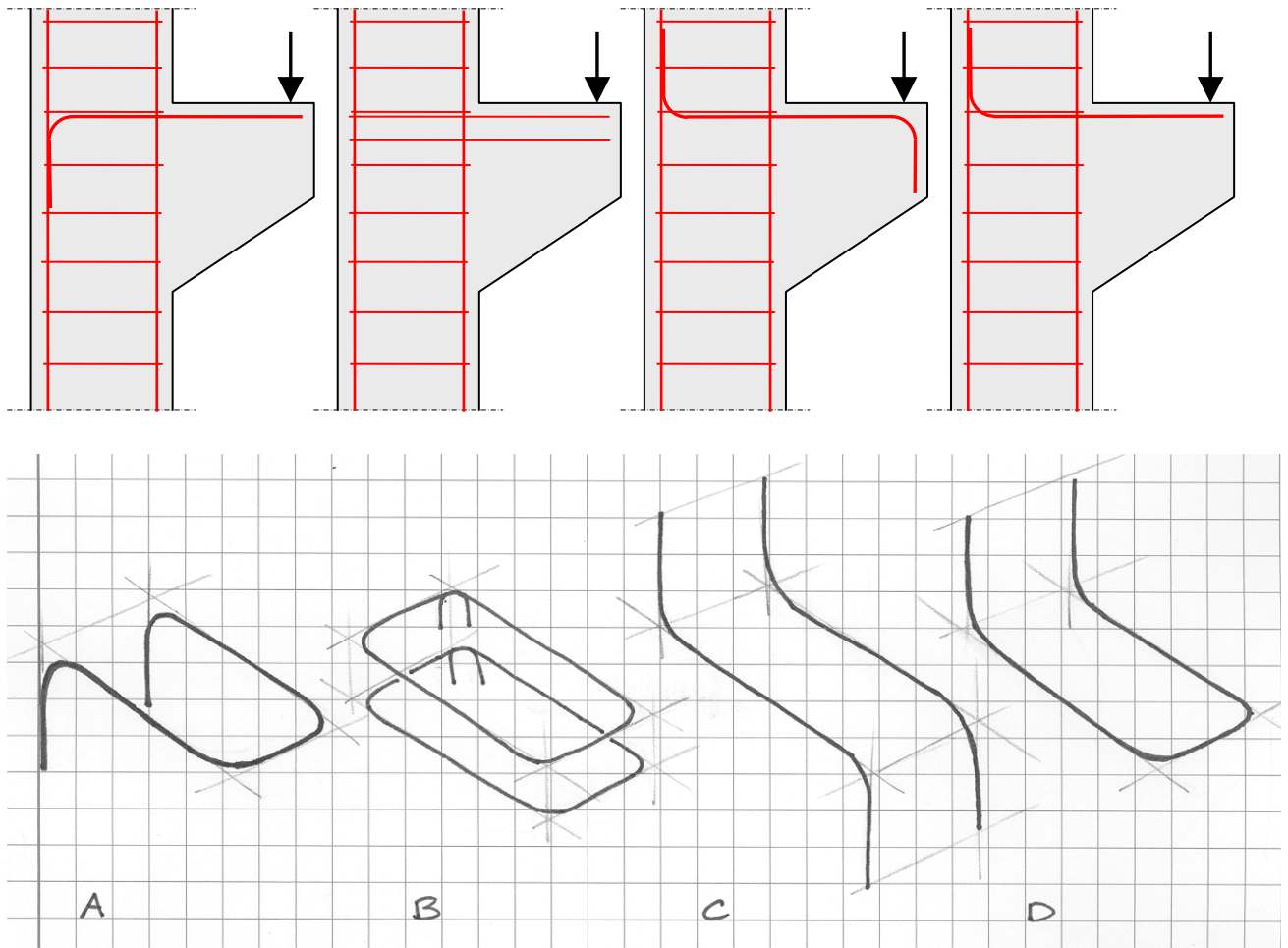
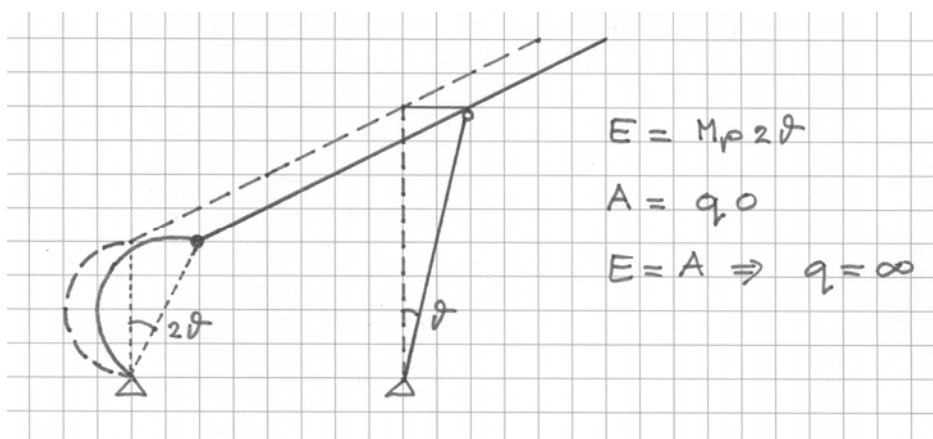
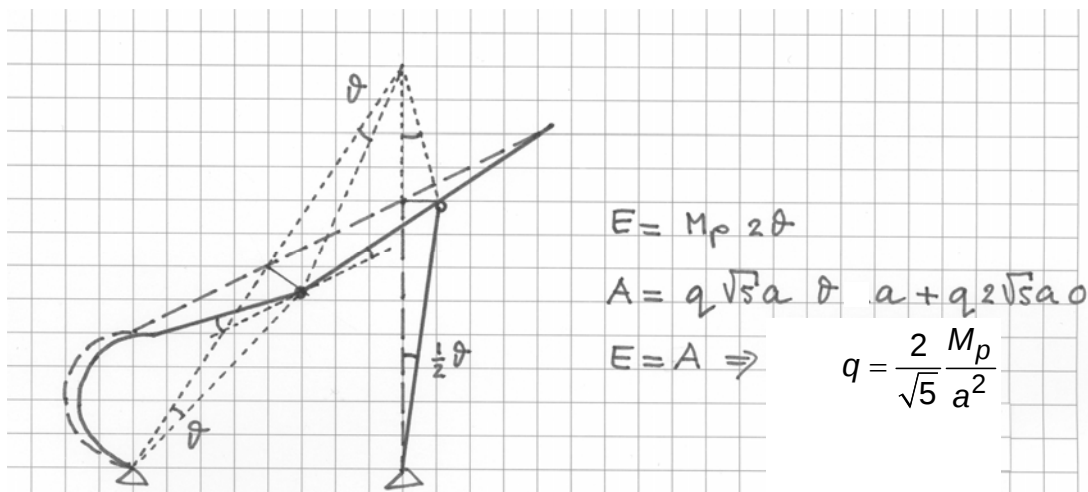
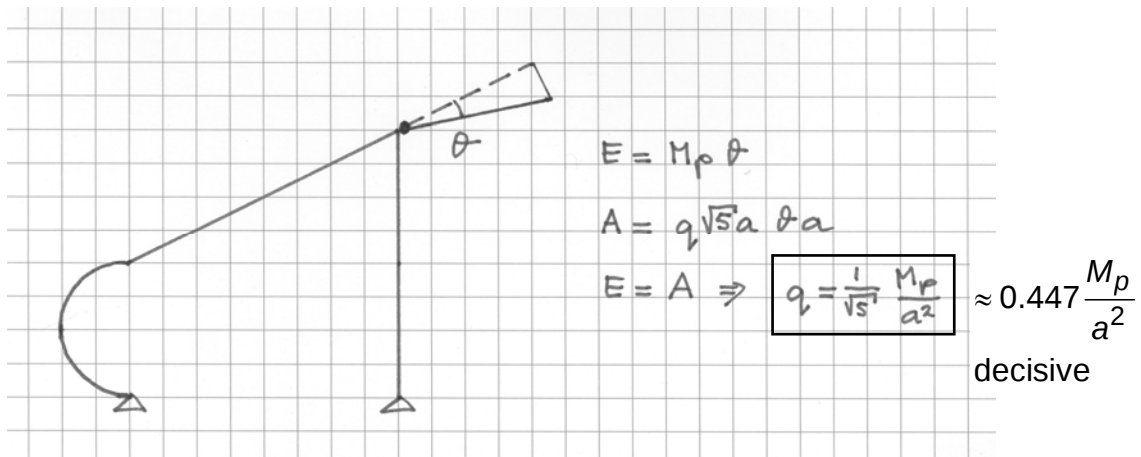
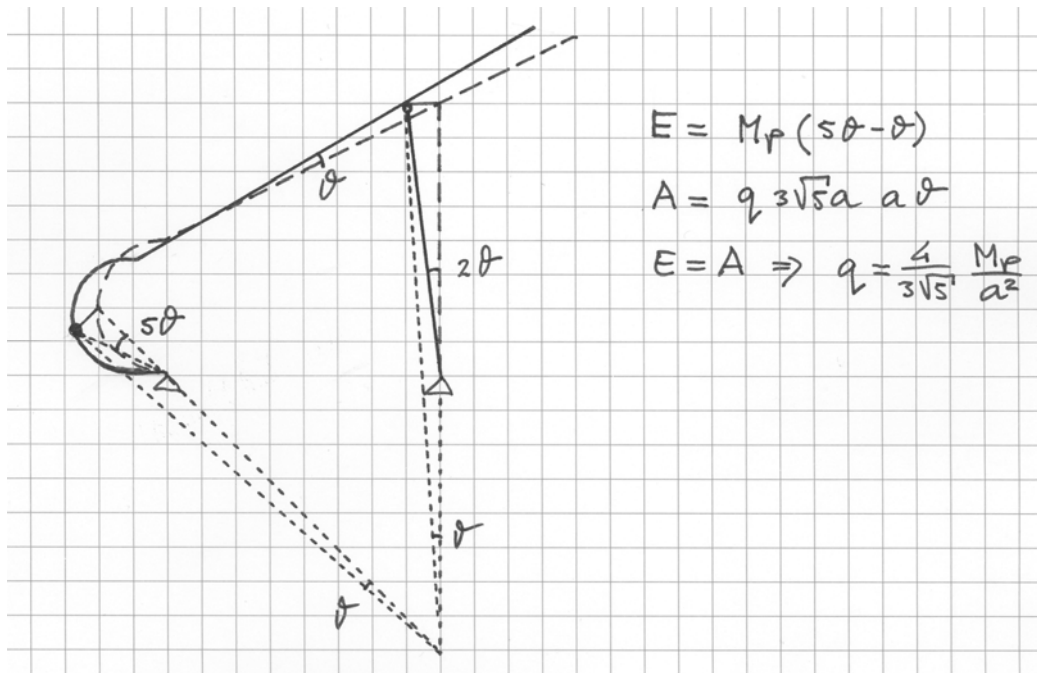


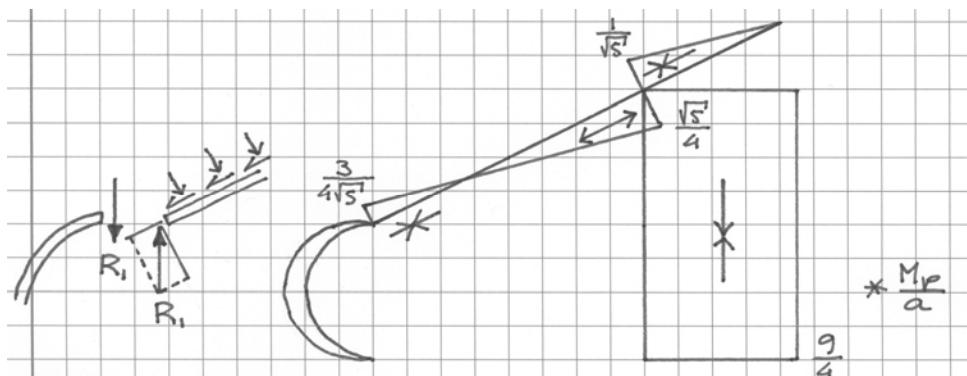
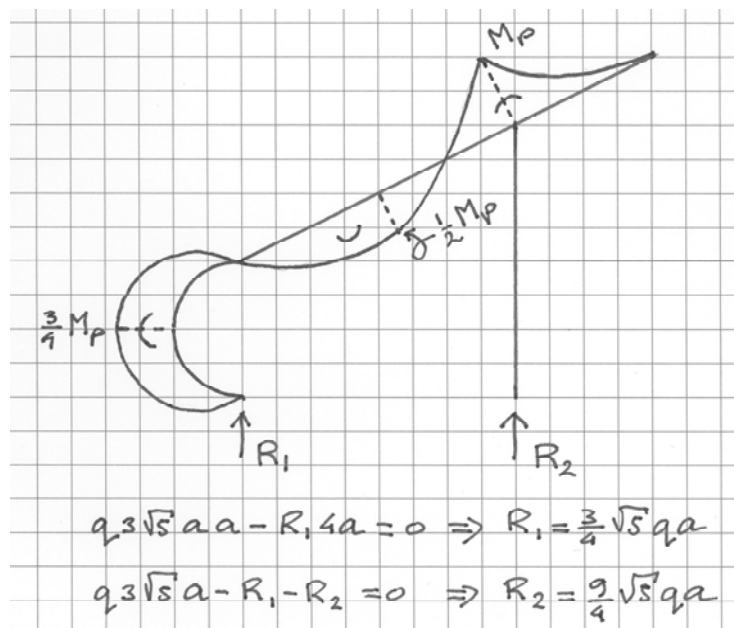
Figure 8. Reinforcement detailing

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Answer to Problem 1a





Answer to Problem 1b



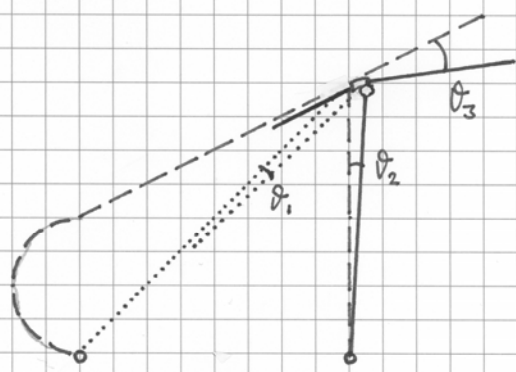
Answer to problem 1c Lower-bound

$$M = \frac{\sqrt{5}}{4} \frac{M_p}{a} \alpha = (1-\alpha) M_p = (1-\alpha) \beta \frac{M_p}{a}$$

$$\alpha \frac{\sqrt{5}}{4} = (1-\alpha) \beta \Rightarrow \alpha = \frac{\beta}{\beta + \frac{\sqrt{5}}{4}} = \frac{120}{121}$$

$$q = \alpha \frac{1}{\sqrt{5}} \frac{M_p}{a^2} = \frac{120}{121} \frac{1}{\sqrt{5}} \frac{M_p}{a^2} \approx 0.444 \frac{M_p}{a^2}$$

Answer to Problem 1c Upper-bound



$u = \theta_2 \frac{a}{\beta}$
 $\beta = 30\sqrt{5}$

hor. displ. $\theta_1 4a + (\theta_3 - \theta_1) \frac{a}{\beta} \frac{2}{\sqrt{5}} = \theta_2 4a$
 vert. displ. $\theta_1 4a - (\theta_3 - \theta_1) \frac{a}{\beta} \frac{1}{\sqrt{5}} = 0$

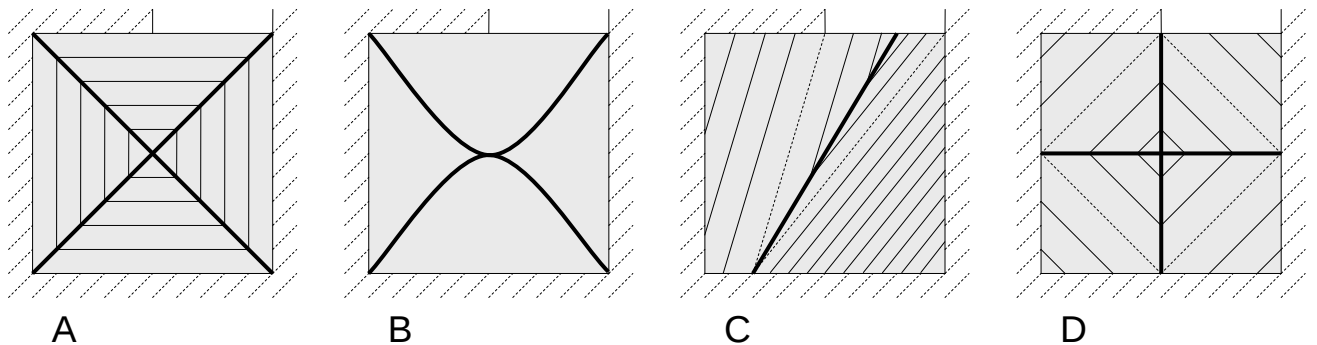
$E = M_p (\theta_3 - \theta_1)$
 $A = q \sqrt{5} a \theta_2 a + q 2\sqrt{5} a \theta_1 2a$
 $E = A$

The solution to the equations is

$$\theta_1 = \frac{1}{601} \theta_3 \quad \theta_2 = \frac{3}{601} \theta_3 \quad q = \frac{24}{121} \sqrt{5} \frac{M_p}{a^2} \approx 0.444 \frac{M_p}{a^2}$$

Answer to Problem 2a

Kinematically possible are pattern A, C, D. The figure below shows the altitude lines of the deformed mechanisms.



Answer to Problem 2b

$$E = 4[\varphi m_p l] = 8 w m_p$$

$$l = a\sqrt{\frac{1}{4} + \alpha^2}$$

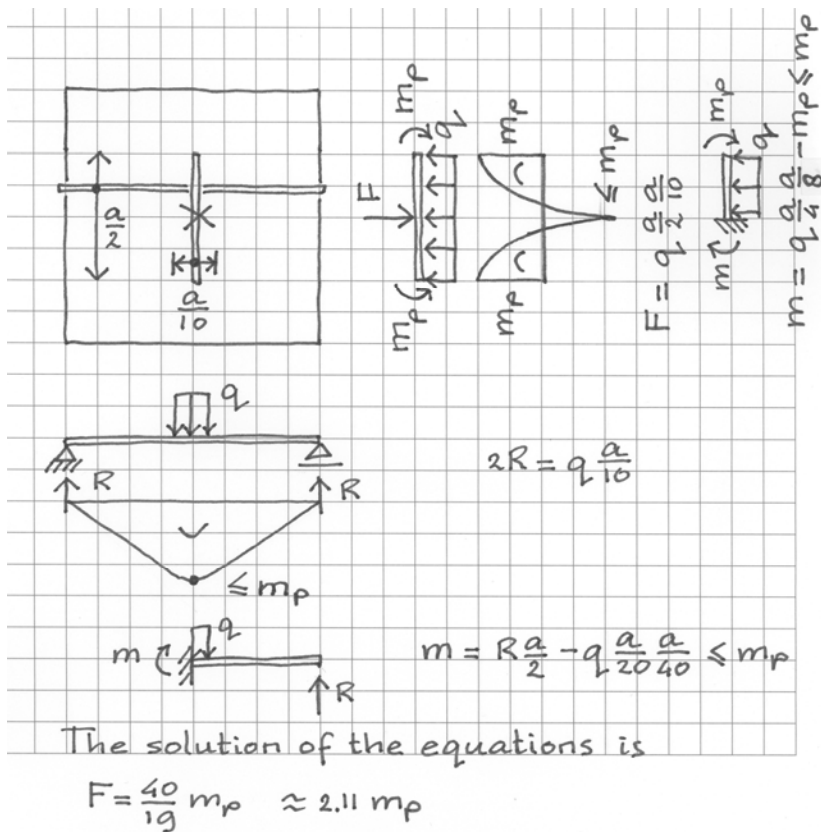
$$\varphi = 2\frac{w}{l}$$

$$A = Fw$$

$$E = A \Rightarrow \boxed{F = 8m_p}$$

Note that the result does not depend on α .

Answer to Problem 2c



Answer to Problem 3a

$$\tan \alpha = \frac{m_{px} - m_{xx}}{m_{xy}} \quad (12.10)$$

$$= \frac{(m_{xx} + |m_{xy}|) - m_{xx}}{m_{xy}} = \frac{|m_{xy}|}{m_{xy}} = \pm 1$$

$$\alpha = 45^\circ \text{ when } m_{xy} \geq 0$$

$$\alpha = -45^\circ \text{ when } m_{xy} \leq 0$$

Answer to Problem 3b

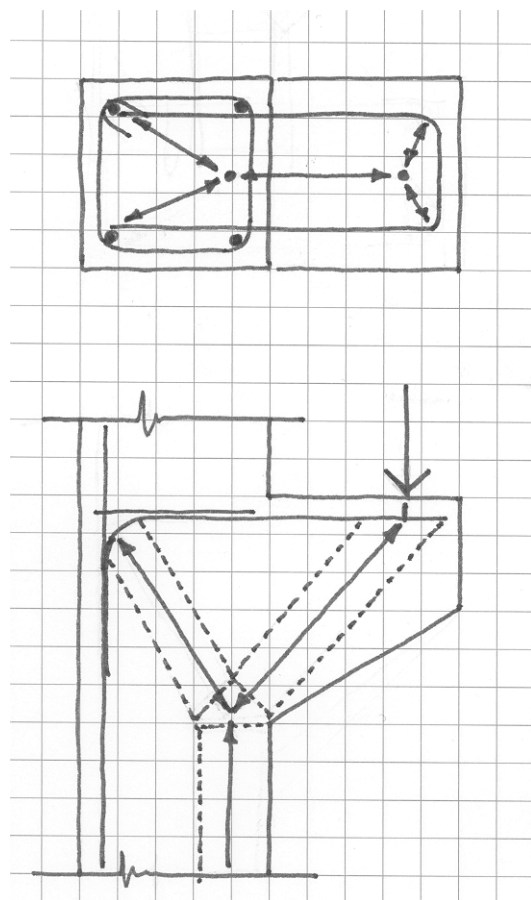
Mid plate (p. 47)	$F = 4\pi m_p \Rightarrow m_p = \frac{F}{12.6}$	safe
Plate edge (p. 50)	$F = (2 + \pi)m_p \Rightarrow m_p = \frac{F}{5.14}$	safe
Plate corner (p. 50)	$F = 2m_p \Rightarrow m_p = \frac{F}{2}$	not safe

Answer to Problem 3c

A or B are correct (see figure)

C will split at the load point

D does not connect properly to the left inclined strut.



Strut-and-tie model for detailing A