

Exam CIE4150 Plastic Analysis of Structures
 Thursday 22 January 2019, 13:30 – 16:30 hours

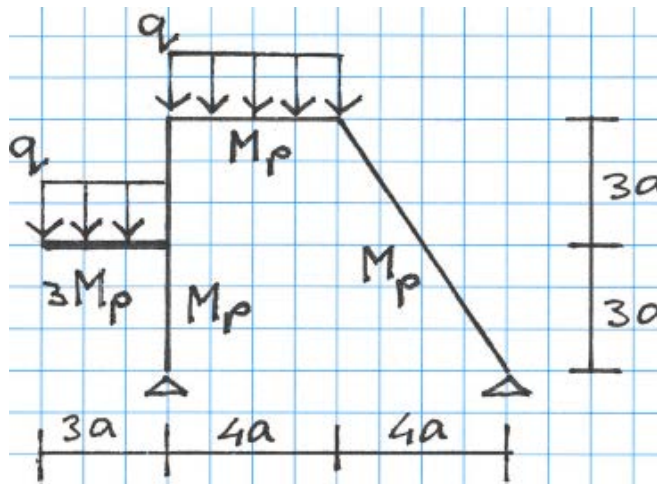


Figure 1. Frame structure

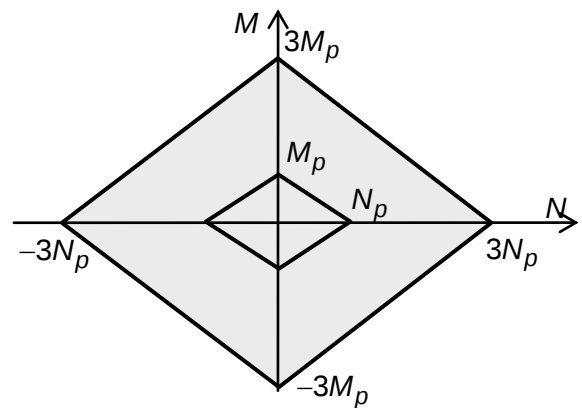


Figure 2. Yield contours

Problem 1

A frame consists of two columns, a beam and a cantilever (Fig.1). The elements have a strength M_p except for the cantilever, which has a strength $3M_p$. The elements are rigidly connected. The support consist of two hinges. The structure is loaded by two evenly distributed line loads q . The relation of Figure 2 exists between the plastic moments and the plastic normal forces.

$$N_p = \beta \frac{M_p}{a}$$

The influence of shear on the yield contour is neglected. Buckling and second order effects are not considered.

- a** Assume $\beta \rightarrow \infty$. Determine the collapse load q for all possible mechanisms. Write the collapse loads as functions of M_p and a . What is the decisive collapse load? (1.5 point)
- b** Assume $\beta \rightarrow \infty$. Draw the bending moment diagram and normal force diagram for the structure at the moment of collapse. (1.5 points)
- c** Assume $\beta = 12$. Choose one of the following problems (You need not do both).
 - Determine the largest lower-bound for q .
 - Determine the smallest upper-bound for q .
 You only need to write down the equations and not solve the equations (1.5 points).

Problem 2

A reinforced concrete plate has simply supported edges and free edges (Fig. 3). It carries an evenly distributed load p [kN/m²]. There is no other load on the plate. The plate is homogeneous and orthotropic.

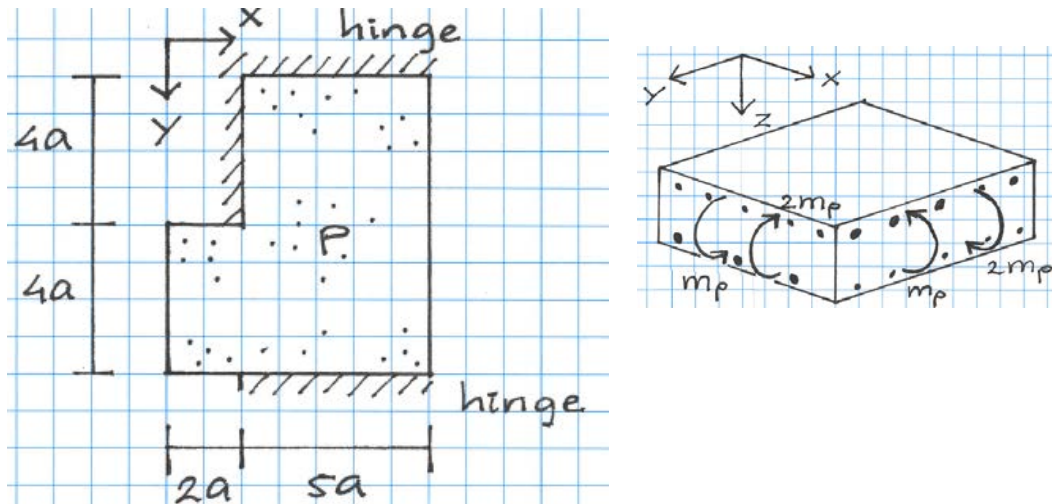


Figure 3. Plate dimensions and reinforcement

- a Consider the yield line patterns of Figure 4. Which of these patterns give kinematically possible mechanisms? (1 point)

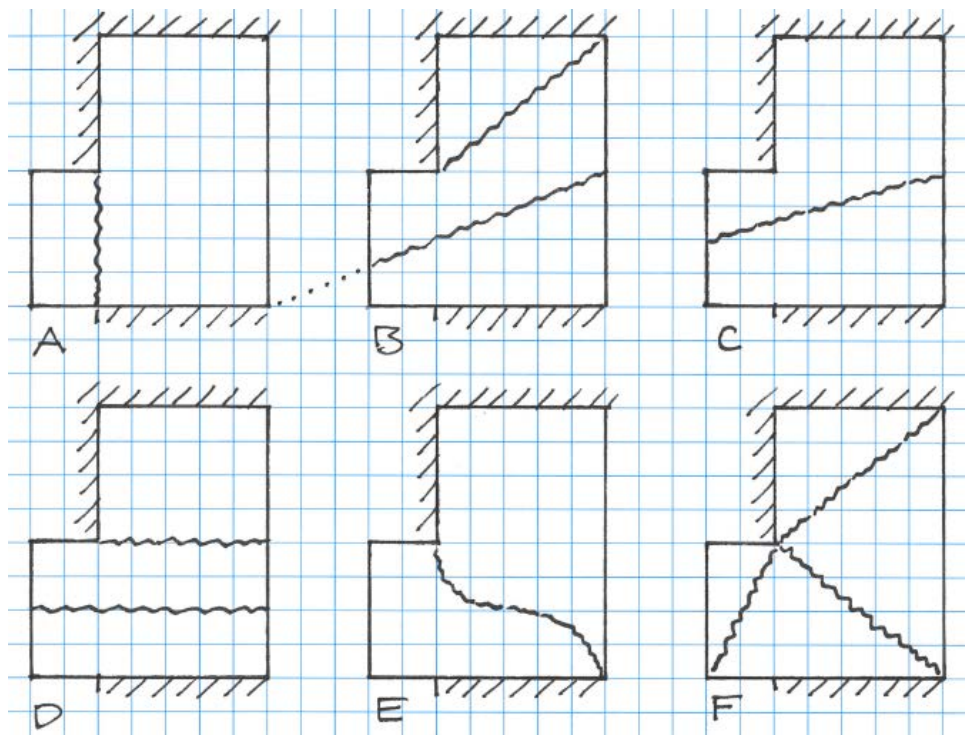


Figure 4. Yield line patterns of problem 2a

- b Consider the yield line pattern of Figure 5. Determine an upper bound for p expressed in m_p and a (1.5 point).

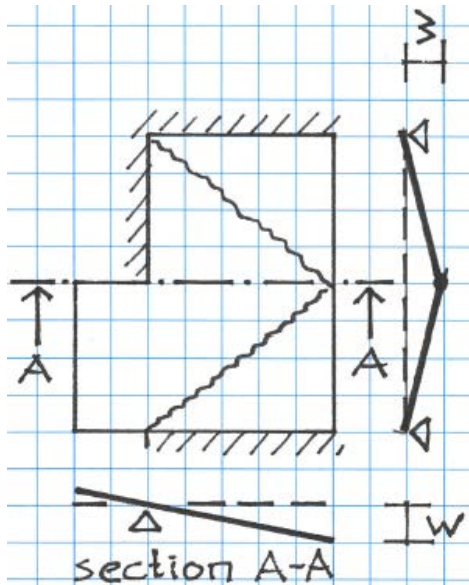


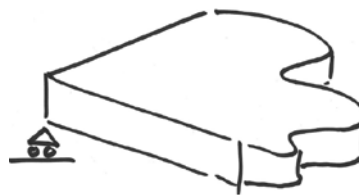
Figure 5. Mechanism of problem 2b

- c Determine the largest lower-bound for p using torsion free beams ($m_{xy} = 0$). You only need to write down the equations and not solve the equations. (1.5 point)

Problem 3

- a A plate is supported in a 90° corner point. The support is a hinge. The support reaction is perpendicular to the plate. The plate strength is m_p in all directions. The collapse load is ... Choose A, B, C or D. (0.5 points)

- A πm_p
- B $2m_p$
- C $\leq 2m_p$
- D 0 because it is a singularity

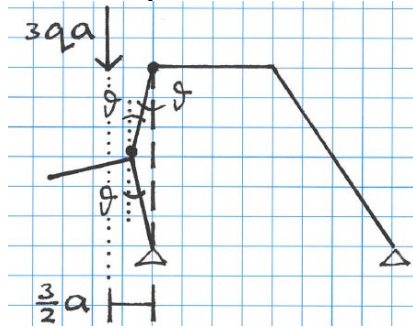


- b In limit analysis, how do we know whether the exact plastic strength has been found? Choose A, B, C or D. (0.5 points)

- A When the lower- and upper-bound are the same.
- B When all possible mechanisms have been considered.
- C When it has been confirmed by finite element analysis.
- D When Prager's theorems can be proved.

- c Which yield contour is most suitable for modelling metals? How do we know this is true? (0.5 points)

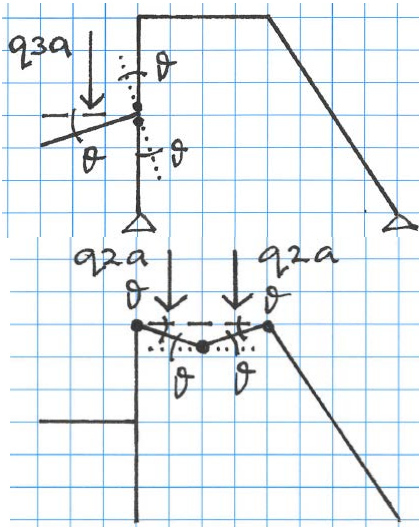
Answer to problem 1a



```
> E:=Mp*t +Mp*(t+t):
> A:=q*3*a*t*3/2*a:
> q:=solve(E=A,q);evalf(q);
```

$$q := \frac{2}{3} \frac{Mp}{a^2}$$

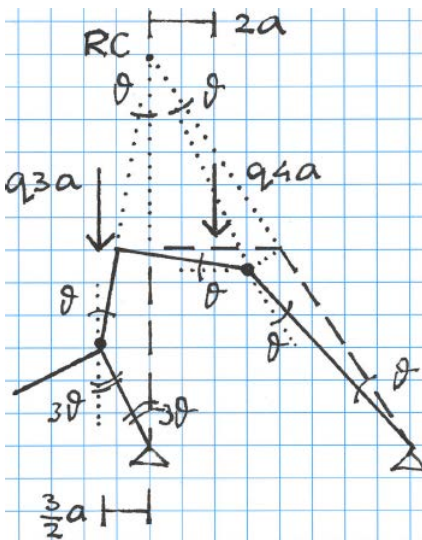
$$\frac{0.6666666667 Mp}{a^2}$$



```
> E:=Mp*t +Mp*t:
> A:=q*3*a*t*3/2*a:
> q:=solve(E=A,q);evalf(q);
```

$$\frac{4}{9} \frac{Mp}{a^2}$$

$$\frac{0.4444444444 Mp}{a^2}$$

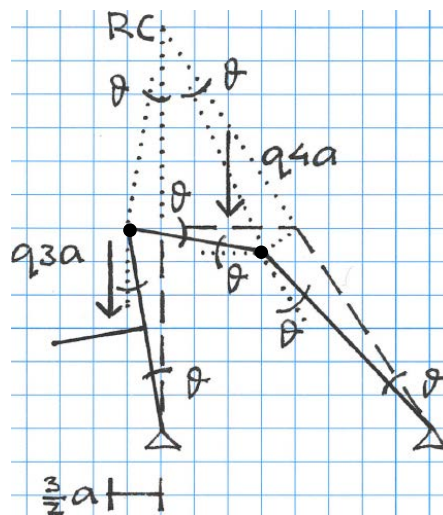


```
> E:=Mp*(3*t+t) +Mp*(t+t):
> A:=q*3*a*3*t*3/2*a +q*4*a*t*2*a:
> q:=solve(E=A,q);evalf(q);
```

$$q := \frac{12}{43} \frac{Mp}{a^2}$$

$$\frac{0.2790697674 Mp}{a^2}$$

decisive

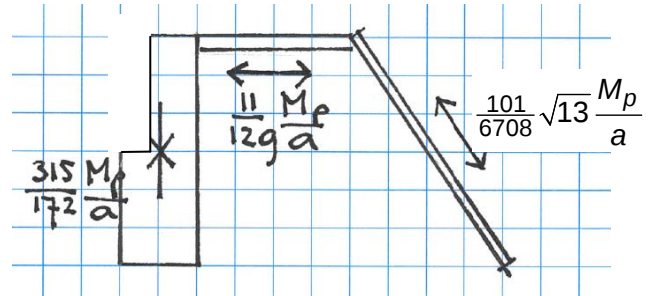
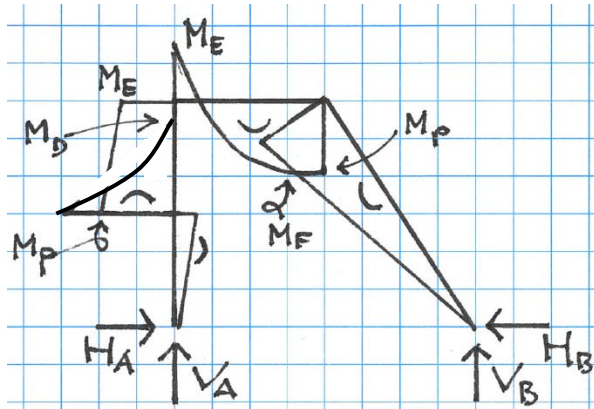


```
> E:=Mp*(t+t) +Mp*(t+t):
> A:=q*3*a*t*3/2*a +q*4*a*t*2*a:
> q:=solve(E=A,q);evalf(q);
```

$$q := \frac{8}{25} \frac{Mp}{a^2}$$

$$\frac{0.3200000000 Mp}{a^2}$$

Answer to problem 1b



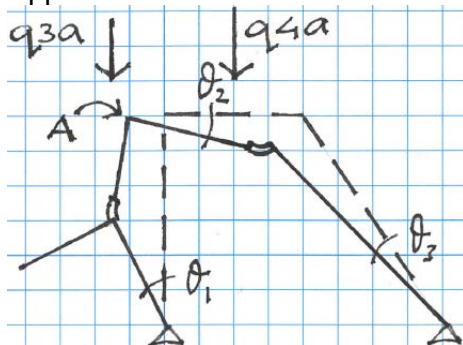
```

> q:=12/43*Mp/a^2:
> eq1:= VA +VB -q*3*a -q*4*a =0:
> eq2:= HA -HB =0:
> eq3:= q*3*a *3/2*a -q*4*a *2*a +VB *8*a=0:
> eq4:= Mp = VB*4*a -HB*6*a:
> opl:=solve({eq1,eq2,eq3,eq4},{VA,VB,HA,HB});
      opl := { HA = - 11/129 * Mp/a, HB = - 11/129 * Mp/a, VA = 315/172 * Mp/a, VB = 21/172 * Mp/a }
> assign(opl):
> MC:= HB*3*a -VB*8*a +q*4*a*2.*a;
      MC := 1.000000000 Mp
> MD:=q*3*a*3/2.*a;
      MD := 1.255813953 Mp
> ME:= q*4*a*2*a -VB*8*a +HB*6.*a;
      ME := 0.7441860470 Mp
> MF:= VB*5*a -HB*6*a -q*a*1/2.*a;
      MF := 0.9825581396 Mp
> l:=sqrt(4^2+6^2):
> N:=VB*6/l+HB*4/l;
      N := 101/6708 * Mp*sqrt(13)/a

```

Answer to problem 1c

Upperbound



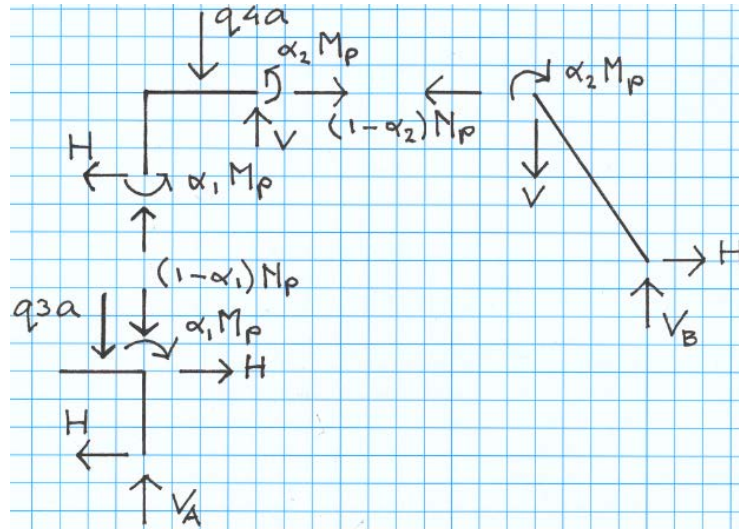
```

> b:=12:
> eq1:= t1*3*a -t2*3*a = t3*6*a +(t2+t3)*a/b:
> eq2:= (t1+t2)*a/b = t3*4*a -t2*4*a:
> E:= Mp*(t1+t2) + Mp*(t2+t3):
> A:= q*3*a *t1*3/2*a +q*4*a *(t3*4*a-t2*2*a):
> solve({eq1,eq2,E=A},{t1,t2,q});

$$\left\{ q = \frac{6976 Mp}{26443 a^2}, t1 = \frac{5353 t3}{1801}, t2 = \frac{1655 t3}{1801} \right\}$$


```

Lowerbound



```

> Np:=b*Mp/a:
> eq1:= VA = q*3*a +(1-a1)*Np:
> eq2:= a1*Mp +H*3*a -q*3*a *3/2*a = 0:
> eq3:= (1-a1)*Np +V = q*4*a:
> eq4:= a2*Mp = (1-a1)*Np*4*a -a1*Mp -q*4*a*2*a + H*3*a:
> eq5:= (1-a2)*Np = H:
> eq6:= a2*Mp = VB*4*a +H*6*a:
> eq7:= V = VB:
> solve({eq1,eq2,eq3,eq4,eq5,eq6,eq7},{q,a1,a2,VA,VB,V,H});

$$\left\{ H = \frac{2340 Mp}{26443 a}, V = \frac{3052 Mp}{26443 a}, VA = \frac{45780 Mp}{26443 a}, VB = \frac{3052 Mp}{26443 a}, a1 = \frac{24372}{26443}, a2 = \frac{26248}{26443}, q = \frac{6976 Mp}{26443 a^2} \right\}$$


```

Answer to problem 2a

A, B, D, F

3 or less correct 0 point
 4 correct 0.3 point
 5 correct 0.7 point
 6 correct 1.0 point

Answer to problem 2b

$$> E := mp \cdot 4 \cdot a \cdot w / (5 \cdot a) + 2 \cdot mp \cdot 5 \cdot a \cdot w / (4 \cdot a) + mp \cdot 4 \cdot a \cdot w / (5 \cdot a) + 2 \cdot mp \cdot 5 \cdot a \cdot w / (4 \cdot a);$$

$$E := \frac{33}{5} mp w$$

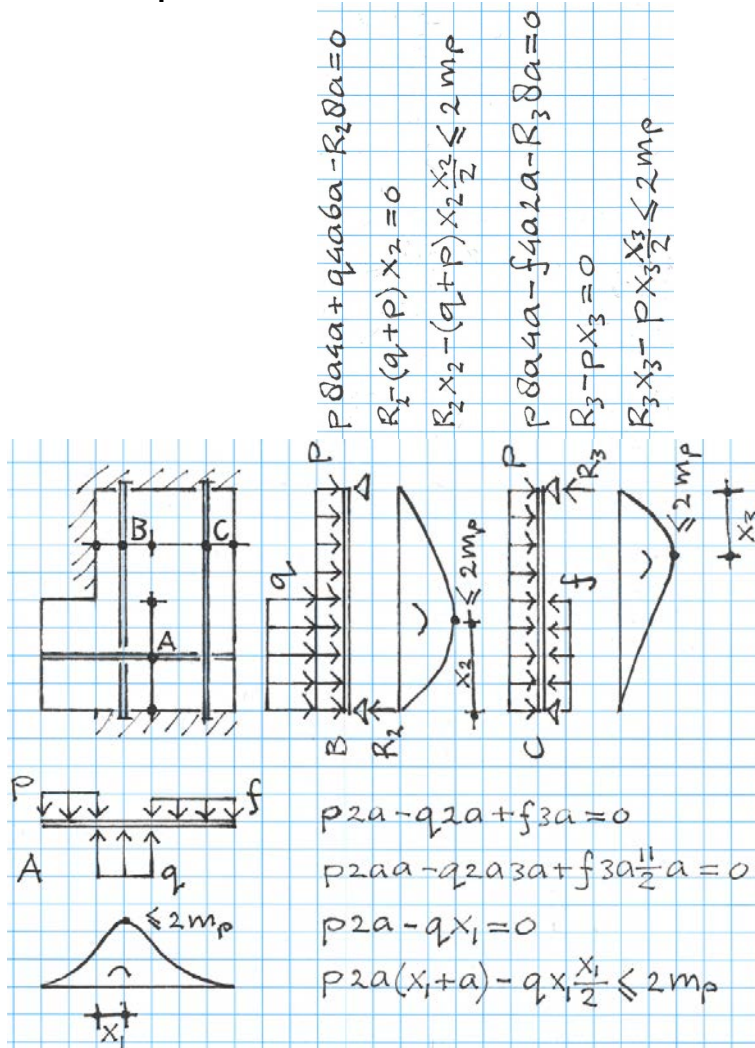
$$> A := 4 \cdot (p \cdot 1/2 \cdot 5 \cdot a \cdot 4 \cdot a \cdot w / 3) - p \cdot 2 \cdot a \cdot 4 \cdot a \cdot w / 5;$$

$$A := \frac{176}{15} p a^2 w$$

$$> p := \text{solve}(E=A, p);$$

$$p := \frac{9}{16} \frac{mp}{a^2}$$

Answer to problem 2c



The solution to the equations is $p = \frac{280}{2209} \frac{mp}{a^2}$.

The latter answer has been computed with the following Maple script.
(Not required for the exam.)

```
> eq1:=p*2*a-q*2*a+f*3*a=0:
> eq2:=p*2*a*a-q*2*a*3*a+f*3*a*11/2*a=0:
> eq3:=p*2*a-q*x1=0:
> eq4:=p*2*a*(x1+a)-q*x1*x1/2=2*mp:
> eq5:=p*8*a*4*a+q*4*a*6*a-R2*8*a=0:
> eq6:=R2-(q+p)*x2=0:
> eq7:=R2*x2-(q+p)*x2*x2/2=2*mp:
> eq8:=p*8*a*4*a-f*4*a*2*a-R3*8*a=0:
> eq9:=R3-p*x3=0:
> eq10:=R3*x3-p*x3*x3/2=2*mp:
> opl:=solve({eq1,eq2,eq3,eq5,eq6,eq7,eq8,eq9},{p,q,f,x1,x2,x3,R2,R3});
      opl := { R2 =  $\frac{56}{47} \frac{mp}{a}$ , R3 =  $\frac{2912}{6627} \frac{mp}{a}$ , f =  $\frac{448}{6627} \frac{mp}{a^2}$ , p =  $\frac{280}{2209} \frac{mp}{a^2}$ , q =  $\frac{504}{2209} \frac{mp}{a^2}$ , x1 =  $\frac{10}{9} a$ , x2 =  $\frac{47}{14} a$ , x3 =  $\frac{52}{15} a$  }
> assign(opl);
> evalf(eq4);
                                0.3943463608 mp = 2. mp
> evalf(eq7);
                                2. mp = 2. mp
> evalf(eq10);
                                0.7616518284 mp = 2. mp
```

Answer to problem 3

- a B
- b A
- c Von Mises, experiments