

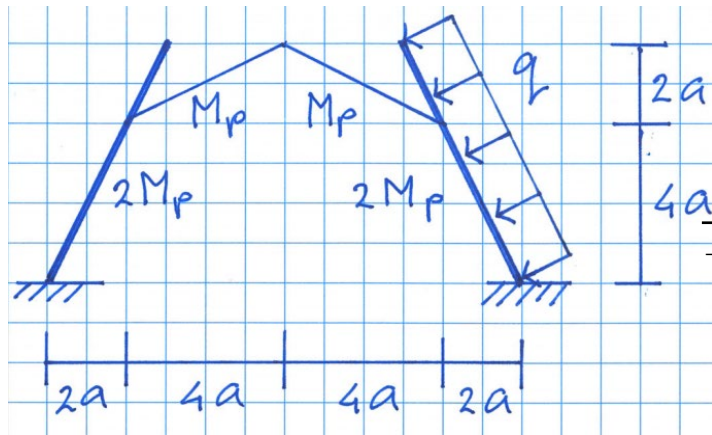


Figure 1. Frame structure

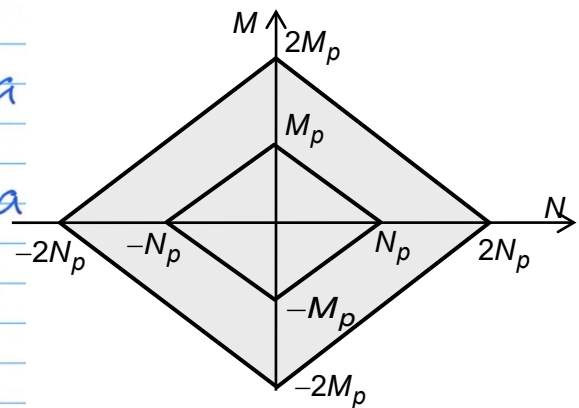


Figure 2. Yield contours

Problem 1

A frame consists of four members (Fig.1). The columns have a strength $2M_p$. The roof members have a strength M_p . The members are rigidly connected. The supports are fixed. The structure is loaded by an evenly distributed line load q per column length (wind load). The relation of Figure 2 exists between the plastic moments and the plastic normal forces.

$$N_p = \beta \frac{M_p}{a}$$

The influence of shear on the yield contour is neglected. Buckling and second order effects are not considered.

- a Assume $\beta \rightarrow \infty$. Determine the collapse load q for all possible mechanisms. Write the collapse loads as functions of M_p and a . What is the decisive collapse load? (1.5 point)
- b Assume $\beta \rightarrow \infty$. Draw the bending moment diagram and normal force diagram for the structure at the moment of collapse. (1.5 points)
- c Assume $\beta = 6$. Choose one of the following problems (You need not do both).
 - Determine the largest lower-bound for q .
 - Determine the smallest upper-bound for q .
You only need to write down the equations and not solve the equations (1.5 points).

Problem 2

A reinforced concrete plate has simply supported edges and free edges (Fig. 3). It carries an evenly distributed load p [kN/m²]. There is no other load on the plate. The plate is homogeneous and orthotropic.

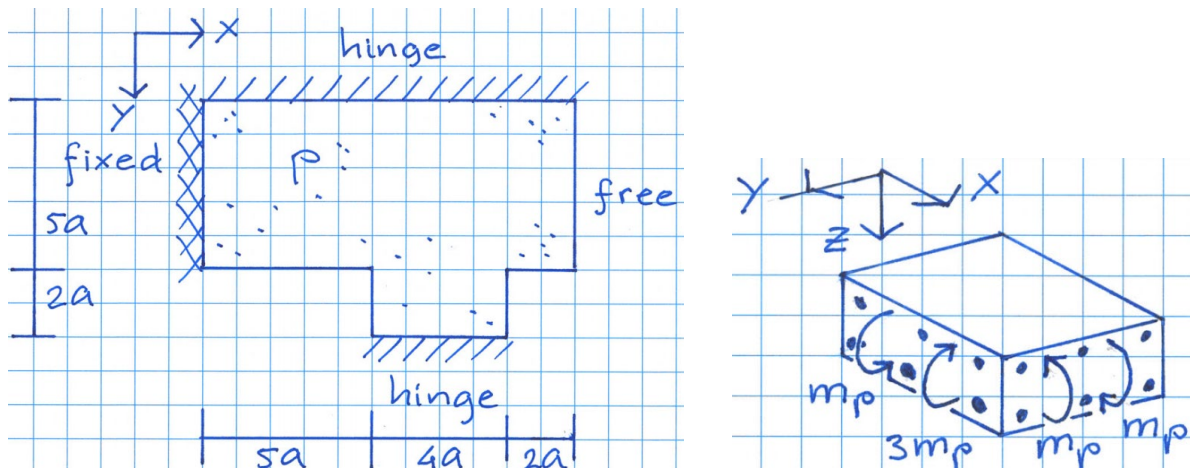


Figure 3. Plate dimensions and reinforcement

- a Consider the yield line patterns of Figure 4. Which of these patterns give kinematically possible mechanisms? (1 point)

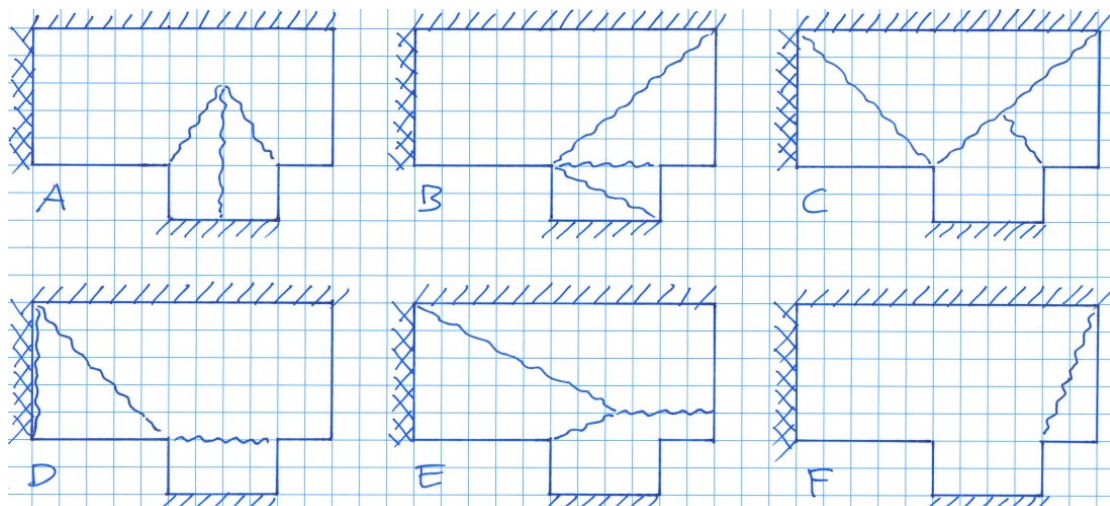


Figure 4. Yield line patterns of problem 2a

- b** Consider the yield line pattern of Figure 5. Determine an upper-bound for p expressed in m_p and a (1.5 point).

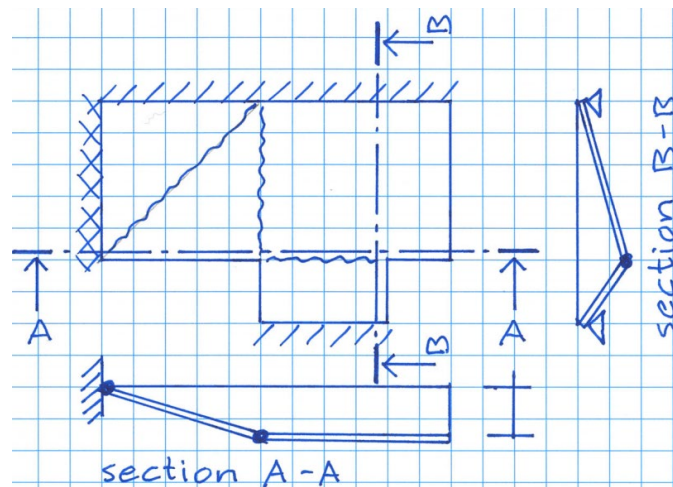


Figure 5. Mechanism of Problem 2b

- c** Determine the largest lower-bound for p using torsion free beams ($m_{xy} = 0$). You only need to write down the equations and not solve the equations. (1.5 point)

Problem 3

- a** What is ductility? Choose A, B, C or D. (0.5 point)
- A The ability to deform with much plastic deformation.
 - B Plastic deformation in the direction of the normal vector on the yield contour.
 - C The material property that the complementary energy remains positive semi-definite.
 - D Shifting of the yield contour during strengthening of yielding.
- b** Welding stresses and rolling stresses reduce the ... Choose A, B, C or D. (0.5 point)
- A ... moment capacity of a beam cross-section.
 - B ... buckling load of a column.
 - C ... deflection of a beam.
 - D ... interaction of plastic moment en normal force in a column cross-section.
- c** The yield lines in a plate transfer bending moments and torsion moments. The torsion moments are neglected in calculating the virtual work. Why? Choose A, B, C or D. (0.5 point)
- A The stresses due to the torsion moments are very small.
 - B The torsion moments must be zero because the material has no extra capacity.
 - C A rhombic interaction diagram is assumed.
 - D There is no torsion deformation in a yield line.

Answer to problem 1a

```
> l:= sqrt(a^2+(2*a)^2):
> E:= 2*Mp*t:
> A:= q*l * 1/2 * t:
> q:= solve(E=A,q); evalf(q);
```

$$q := \frac{4 Mp}{5 a^2}$$

$$\frac{0.8000000000 Mp}{a^2}$$

```
> E:= 2*Mp*t + 2*Mp*(t+t) + 2*Mp*t:
> A:= 2*( q*l * 1/2 * t):
> q:= solve(E=A,q); evalf(q);
```

$$q := \frac{8 Mp}{5 a^2}$$

$$\frac{1.6000000000 Mp}{a^2}$$

```
> E:= Mp*t + 2*Mp*(t+t) + 2*Mp*t:
> A:= q*l*1/2*t + q*l*1/2*t - q*l*1/2*t:
> q:= solve(E=A,q); evalf(q);
```

$$q := \frac{14 Mp}{5 a^2}$$

$$\frac{2.8000000000 Mp}{a^2}$$

```
> E:= 2*Mp*2*t + Mp*(2*t+t) + Mp*(2*t+t) + 2*Mp*2*t:
> A:= q*3*l * 3/2*1 *2t:
> q:= solve(E=A,q); evalf(q);
```

$$q := \frac{14 Mp}{45 a^2}$$

$$\frac{0.3111111111 Mp}{a^2}$$

```
> E:= 2*Mp*t + Mp*(t+3*t) + Mp*(3*t+3*t) + 2*Mp*3*t:
> A:= q*3*l * 3/2*1 * 3*t:
> q:= solve(E=A,q); evalf(q);
```

decisive

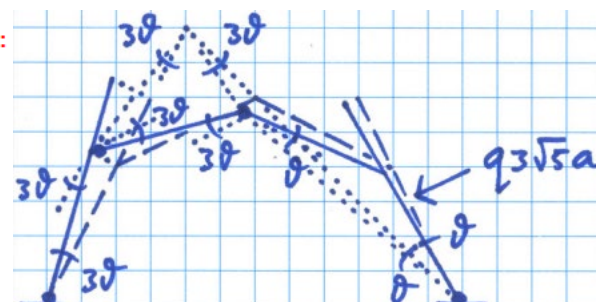
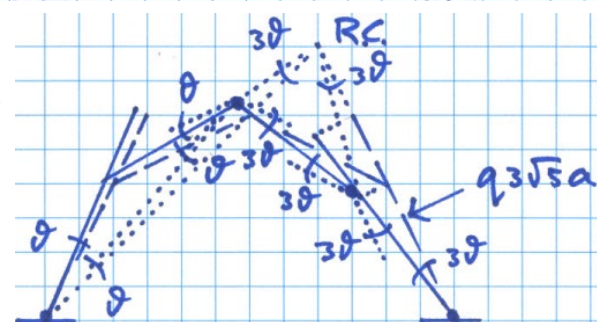
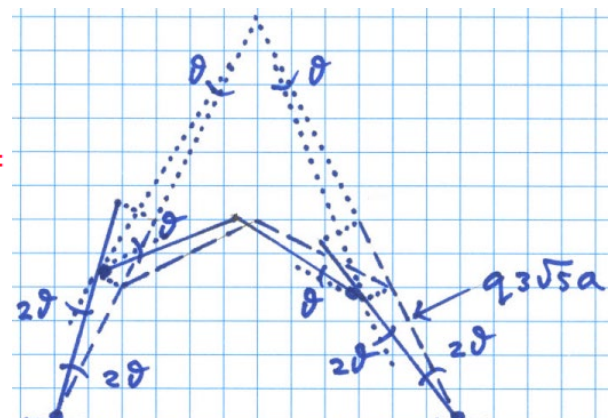
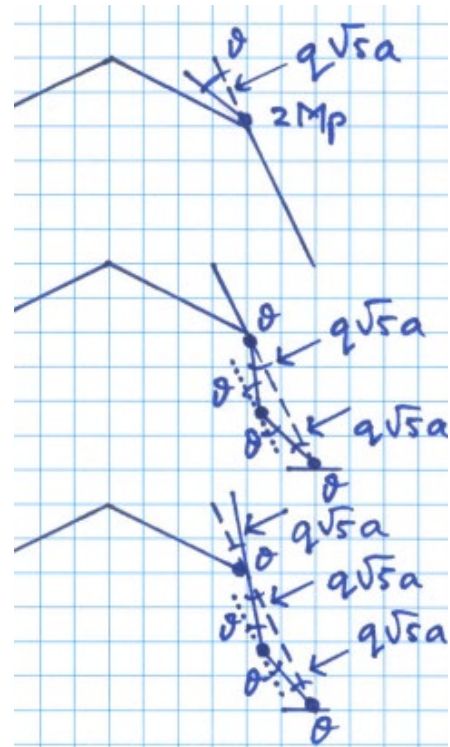
$$q := \frac{4 Mp}{15 a^2}$$

$$\frac{0.2666666667 Mp}{a^2}$$

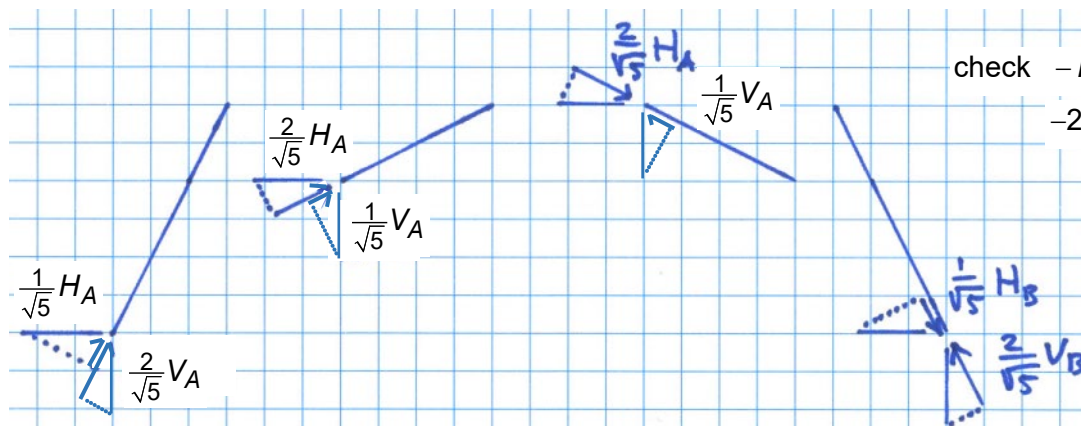
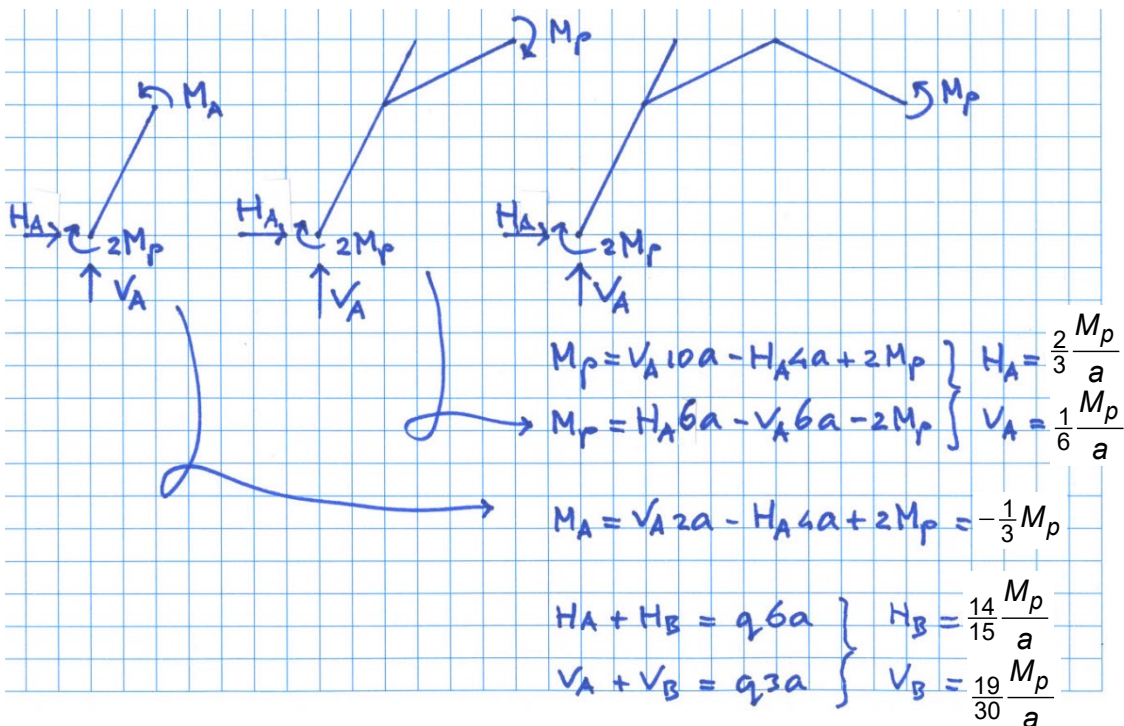
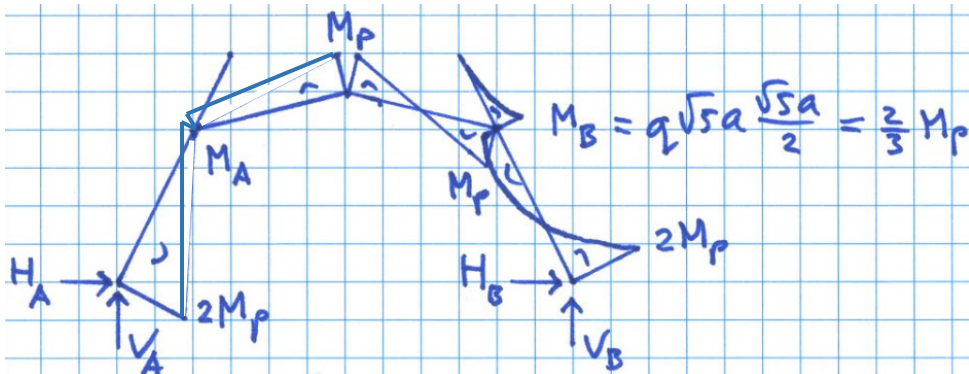
```
> E:= 2*Mp*3*t + Mp*(3*t+3*t) + Mp*(3*t+t) + 2*Mp*t:
> A:= q*3*l * 3/2*1 * t:
> q:= solve(E=A,q); evalf(q);
```

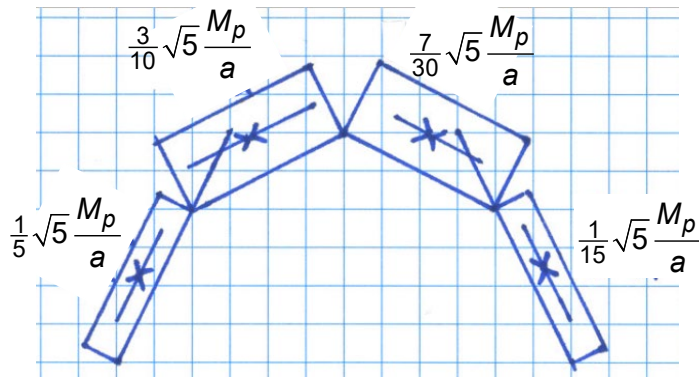
$$q := \frac{4 Mp}{5 a^2}$$

$$\frac{0.8000000000 Mp}{a^2}$$



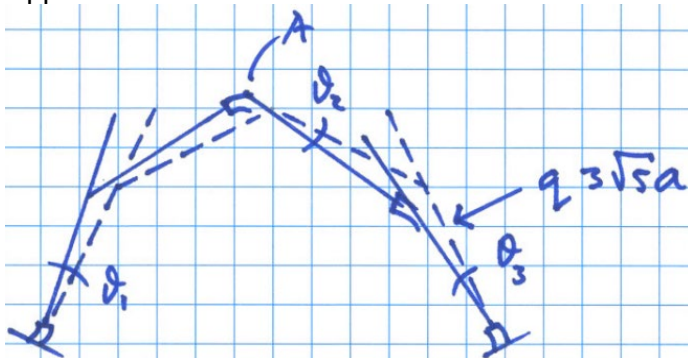
Answer to problem 1b





Answer to problem 1c

Upperbound



> b:=6:

> A_hor_left:= t1*6*a + t1*a/b*1/sqrt(5) + (t1+t2)*a/b*2/sqrt(5) :

> A_vert_left:= t1*6*a - t1*a/b*2/sqrt(5) - (t1+t2)*a/b*1/sqrt(5) :

> A_hor_right:= t3*4*a - t3*a/b*1/sqrt(5) - t2*2*a - (t2+t3)*a/b*2/sqrt(5) :

> A_vert_right:= -t3*2*a - t3*a/b*2/sqrt(5) + t2*4*a - (t2+t3)*a/b*1/sqrt(5) :

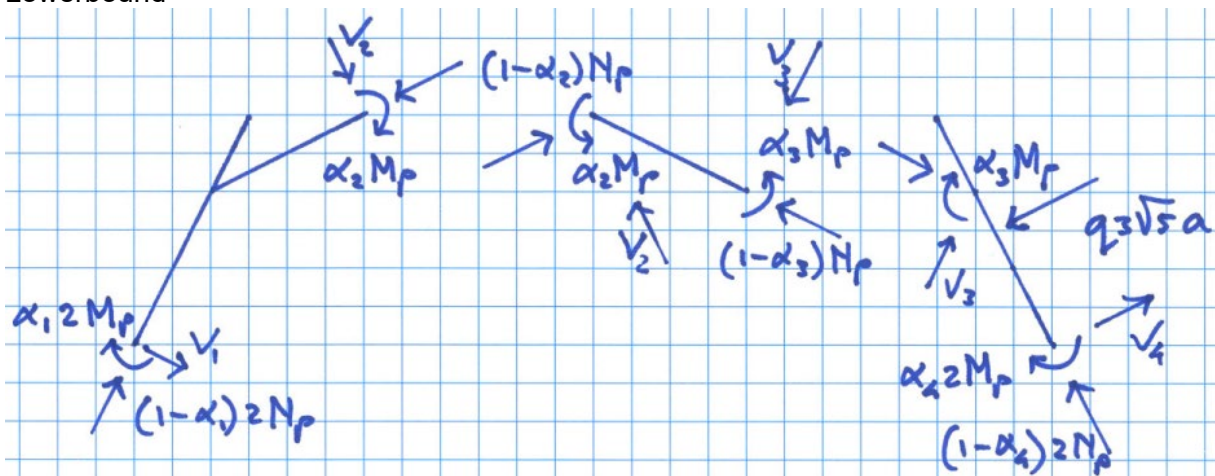
> E:= 2*Mp*t1 + Mp*(t1+t2) + Mp*(t2+t3) + 2*Mp*t3:

> A:= q*3*sqrt(5.)*a * 3*sqrt(5)*a/2 * t3:

> solve({A_hor_left=A_hor_right,A_vert_left=A_vert_right,E=A},{q,t2,t3});

$$\left\{ q = \frac{0.2512774835 M_p}{a^2}, t_2 = 3.566916657 t_1, t_3 = 3.818693770 t_1 \right\}$$

Lowerbound



```

> b:=6: Np:=b*Mp/a:
> eq1:= V1*2/sqrt(5.) + (1-a1)*2*Np*1/sqrt(5) + V2*1/sqrt(5) - (1-a2)*Np*2/sqrt(5) =0:
> eq2:= V1*1/sqrt(5) - (1-a1)*2*Np*2/sqrt(5) + V2*2/sqrt(5) + (1-a2)*Np*1/sqrt(5) =0:
> eq3:= a1*2*Mp = (1-a2)*Np*2/sqrt(5)*6*a - (1-a2)*Np*1/sqrt(5)*6*a - V2*1/sqrt(5)*6*a - V2*
2/sqrt(5)*6*a - a2*Mp:
>
> eq4:= (1-a2)*Np*2/sqrt(5) - V2*1/sqrt(5) - (1-a3)*Np*2/sqrt(5) - V3*1/sqrt(5) =0:
> eq5:=-(1-a2)*Np*1/sqrt(5) - V2*2/sqrt(5) - (1-a3)*Np*1/sqrt(5) + V3*2/sqrt(5) =0:
> eq6:= a2*Mp = V3*2*sqrt(5)*a - a3*Mp:
>
> eq7:= (1-a3)*Np*2/sqrt(5) + V3*1/sqrt(5) - (1-a4)*2*Np*1/sqrt(5) + V4*2/sqrt(5) - q*6*a =0:
> eq8:= (1-a3)*Np*1/sqrt(5) - V3*2/sqrt(5) - (1-a4)*2*Np*2/sqrt(5) - V4*1/sqrt(5) + q*3*a =0:
> eq9:= a3*Mp = V4*2*sqrt(5)*a - q*3*sqrt(5)*a*1/2*sqrt(5)*a - a4*2*Mp:
>
> solve({eq1,eq2,eq3,eq4,eq5,eq6,eq7,eq8,eq9},{q,a1,a2,a3,a4,V1,V2,V3,V4});

$$\left\{ V1 = \frac{0.4868551869 Mp}{a}, V2 = -\frac{0.1458383667 Mp}{a}, V3 = \frac{0.4060763047 Mp}{a}, V4 = \frac{1.068646448 Mp}{a}, a1 = 0.9661603079, a2 \right.$$


$$\left. = 0.8971709740, a3 = 0.9188574688, a4 = 0.9878468028, q = \frac{0.2512774836 Mp}{a^2} \right\}$$


```

Answer to problem 2a

B, D, F

3 or less correct 0.0 point
 4 correct 0.3 point
 5 correct 0.7 point
 6 correct 1.0 point

Answer to problem 2b

```

> E:= p*5*a*5*a/2*w/3 + p*5*a*6*a*w/2 + p*2*a*4*a*w/2;

$$E := \frac{139}{6} p a^2 w$$

> A:= mp*5*a*w/(5*a) + mp*5*a*w/(5*a) + mp*5*a*w/(5*a) + 3*mp*4*a*(w/(2*a)+w/(5*a));

$$A := \frac{57}{5} mp w$$

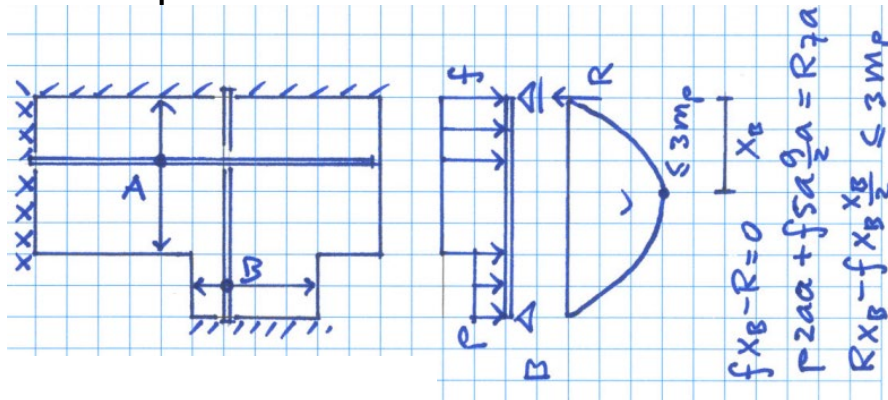
> p:=solve(E=A,p); evalf(p);

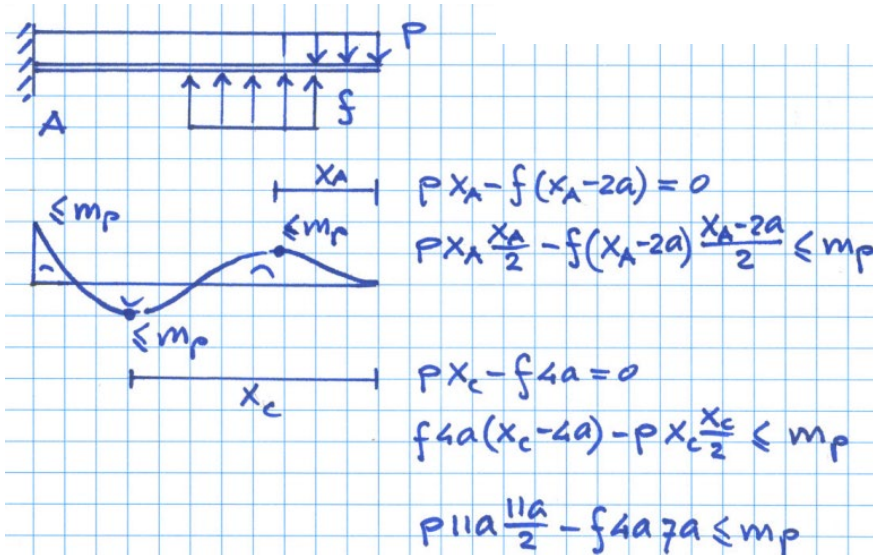
$$p := \frac{342}{695} \frac{mp}{a^2}$$


$$\frac{0.4920863309 mp}{a^2}$$


```

Answer to problem 2c





```

> eq1 := f*xB-R=0:
> eq2 := p*2.*a*a+f*5*a*9/2*a=R*7*a:
> eq3 := R*xB-f*xB*xB/2=3*mp:
> eq4 := p*xA-f*(xA-2*a)=0:
> eq5 := p*xA*xA/2-f*(xA-2*a)*(xA-2*a)/2=mp:
> eq6 := p*xC-f*4*a=0:
> eq7 := f*4*a*(xC-4*a)-p*xC*xC/2<mp:
> eq8 := p*11*a*11*a/2-f*4*a*7*a<mp:
> op1:=solve({eq1,eq2,eq3,eq4,eq5,eq6},{R,xA,xB,xC,p,f}); assign(op1):
op1 := {R = 1.789727914 mp/a, f = 0.5338543344 mp/a^2, p = 0.2581864372 mp/a^2, xA
        = 3.873170143 a, xB = 3.352464893 a, xC = 8.270834675 a}
> evalf(eq7);
0.289172532 mp < mp
> evalf(eq8);
0.67235809 mp < mp

```

Answer to problem 3

- a A
- b B
- c B (D is not correct for yield lines that are inclined to the reinforcement directions: the bending rotation in one direction is the torsion rotation in the other.)

